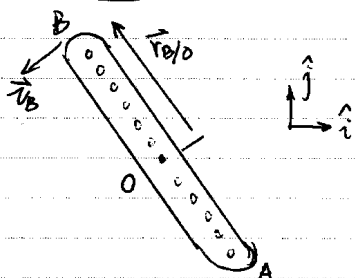


7.57



Suppose the rod is pegged at point O,
and the angular velocity of the rod is $\omega \hat{k}$.

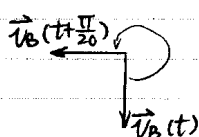
The distance $|OB|$ gives the location of the peg.

From $\vec{v} = \vec{\omega} \times \vec{r}$

$$\text{thus, } \vec{v}_B = \vec{\omega} \times \vec{r}_{B/O} = \omega \hat{k} \times \vec{r}_{B/O}$$

Since, at some instant t , $\vec{v}_B = -3 \text{ m/s } \hat{j}$

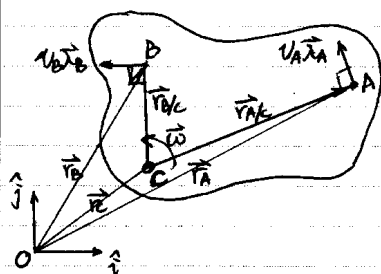
After $\frac{\pi}{20} \text{ s}$, $\vec{v}_B = 3 \text{ m/s } \hat{i}$



$$\therefore \omega = \frac{\Delta \theta}{\Delta t} = \frac{\frac{3}{2}\pi}{\frac{\pi}{20}} = 30 \text{ rad/s}$$

$$|v_B| = \omega \cdot |OB| \Rightarrow |OB| = \frac{|v_B|}{\omega} = \frac{3}{30} = 0.1 \text{ m}$$

7.66.



planar motion
 $\hat{r}_A, \hat{r}_B, \hat{r}_C, \hat{v}_A, \hat{v}_B$ given

a). As shown in figure, suppose the angular velocity is $\omega \hat{k}$. Since it's a rigid body's circular motion, $\vec{\omega}$ is same for whole body.

$$v_A \hat{\lambda}_A = \omega \times (\vec{r}_A - \vec{r}_C)$$

$$v_B \hat{\lambda}_B = \omega \times (\vec{r}_B - \vec{r}_C)$$

+ scalar eqns but 5 unknowns
 $v_A, v_B, \vec{r}_C, \omega$

but we're asked only to find $\vec{r}_C$, thus

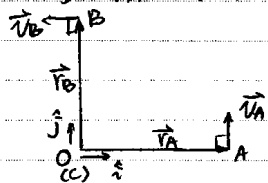
$$\frac{v_A}{\omega} \hat{\lambda}_A = \hat{k} \times (\vec{r}_A - \vec{r}_C)$$

$$\frac{v_B}{\omega} \hat{\lambda}_B = \hat{k} \times (\vec{r}_B - \vec{r}_C)$$

Now 4 unknowns, so the solution to these will yield $\vec{r}_C$.

Illustration:

$$1) \vec{r}_A = 1\hat{i}, \vec{r}_B = 1\hat{j}, \vec{\lambda}_A = 1\hat{j}, \vec{\lambda}_B = -1\hat{i}, \vec{r}_C = \vec{0}$$

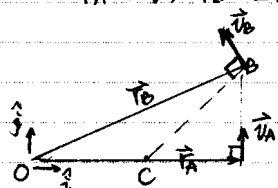


$$\vec{\lambda}_A = \frac{\hat{k} \times (1\hat{i} - \vec{0})}{\|1\hat{i} - \vec{0}\|} = 1\hat{j}$$

$$\vec{\lambda}_B = \frac{\hat{k} \times (1\hat{j} - \vec{0})}{\|1\hat{j} - \vec{0}\|} = -1\hat{i}$$

verified.

$$2) \vec{r}_A = 2\hat{i}, \vec{r}_B = 2\hat{i} + 1\hat{j}, \vec{\lambda}_A = 1\hat{j}, \vec{\lambda}_B = -\frac{\sqrt{5}}{2}\hat{i} + \frac{\sqrt{5}}{2}\hat{j}, \vec{r}_C = 1\hat{i}$$



$$\vec{\lambda}_A = \frac{\hat{k} \times (2\hat{i} - 1\hat{i})}{\|2\hat{i} - 1\hat{i}\|} = \hat{j}$$

$$\vec{\lambda}_B = \frac{\hat{k} \times (2\hat{i} + 1\hat{j} - 1\hat{i})}{\|2\hat{i} + 1\hat{j} - 1\hat{i}\|} = -\frac{1}{\sqrt{2}}\hat{i} + \frac{1}{\sqrt{2}}\hat{j}$$

verified.

b) Point C is located where the lines perpendicular to $\vec{\lambda}_A$ & $\vec{\lambda}_B$ cross.

$$\vec{r}_{B/C} \cdot \vec{\lambda}_B = 0 \Rightarrow (\vec{r}_B - \vec{r}_C) \cdot \vec{\lambda}_B = 0$$

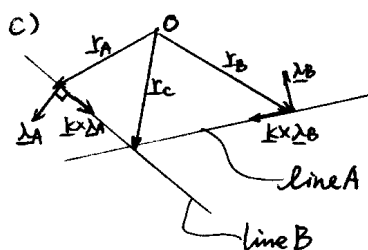
$$\vec{r}_{A/C} \cdot \vec{\lambda}_A = 0 \quad (\vec{r}_A - \vec{r}_C) \cdot \vec{\lambda}_A = 0$$

$$\Rightarrow \begin{cases} \vec{r}_C \cdot \vec{\lambda}_B = \vec{r}_B \cdot \vec{\lambda}_B \\ \vec{r}_C \cdot \vec{\lambda}_A = \vec{r}_A \cdot \vec{\lambda}_A \end{cases}$$

%Problem 7.66 b)

```
%input four given vectors
%should be in a form [x;y]
r_A=input('enter r_A coordinates: ');
lambda_A=input('enter lambda_A coordinates: ');
lambda_A=lambda_A';
r_B=input('enter r_B coordinates: ');
lambda_B=input('enter lambda_B coordinates: ');
lambda_B=lambda_B';
```

```
%Derived from the governing Eqn
A=[ lambda_A;lambda_B];
b=[lambda_A*r_A ;lambda_B*r_B];
x=A\b; % solve linear eqn
disp('r_C is :');
disp(x);
```



parametric eqn. of line A

$$\vec{r}_1 = \vec{r}_A + s(\hat{k} \times \vec{\lambda}_A)$$

parametric eqn. of line B

$$\vec{r}_2 = \vec{r}_B + t(\hat{k} \times \vec{\lambda}_B)$$

At intersection at C: $\mathbf{r}_1 = \mathbf{r}_2$

$$\{\mathbf{r}_A + s(\mathbf{k} \times \boldsymbol{\Delta}_A) = \mathbf{r}_B + t(\mathbf{k} \times \boldsymbol{\Delta}_B)\}$$

Solve for either s or t

$$\{\} \cdot \boldsymbol{\Delta}_B \Rightarrow \mathbf{r}_A \cdot \boldsymbol{\Delta}_B + s \boldsymbol{\Delta}_B \cdot (\mathbf{k} \times \boldsymbol{\Delta}_A) = \boldsymbol{\Delta}_B \cdot \mathbf{r}_B$$

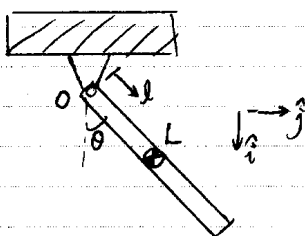
$$\Rightarrow s = \frac{\boldsymbol{\Delta}_B \cdot (\mathbf{r}_B - \mathbf{r}_A)}{\boldsymbol{\Delta}_B \cdot (\mathbf{k} \times \boldsymbol{\Delta}_A)} = \frac{\boldsymbol{\Delta}_B \cdot (\mathbf{r}_B - \mathbf{r}_A)}{\mathbf{k} \cdot (\boldsymbol{\Delta}_A \times \boldsymbol{\Delta}_B)}$$

$$\Rightarrow \mathbf{r}_C = \mathbf{r}_A + s \mathbf{k} \times \boldsymbol{\Delta}_A$$

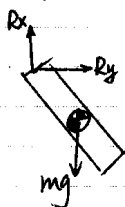
$$\mathbf{r}_C = \mathbf{r}_A + \frac{\boldsymbol{\Delta}_A \cdot (\mathbf{r}_B - \mathbf{r}_A)}{\mathbf{k} \cdot (\boldsymbol{\Delta}_A \times \boldsymbol{\Delta}_B)} \underbrace{\mathbf{k} \times \boldsymbol{\Delta}_A}_{\text{unit vector } \perp \text{ to } \boldsymbol{\Delta}_A \text{ sin(angle bet } \boldsymbol{\Delta}_A \text{ \& } \boldsymbol{\Delta}_B)}}$$

proj. of $\mathbf{r}_{B/A}$ in $\boldsymbol{\Delta}_A$ direction.

7.92



a) FBD:



b) AMB:

$$\boxed{\sum \vec{M}_O = \vec{H}_O}$$

c) left-hand-side:

$$\begin{aligned} \sum \vec{M}_O &= \vec{r}_{O/G} \times m\mathbf{g} (+\hat{i}) \\ &= \frac{1}{2} (\cos\theta \hat{i} + \sin\theta \hat{j}) \times mg \hat{j} \\ &= \boxed{-\frac{1}{2} mgL \sin\theta \hat{k}} \end{aligned}$$

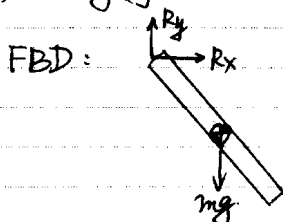
d) right-hand-side:

$$\begin{aligned} \vec{H}_O &= \int_0^L \vec{r}_O \times \vec{a} \frac{m}{L} dl \\ &= \frac{m}{L} \int_0^L l (\cos\theta \hat{i} + \sin\theta \hat{j}) \times [-l\ddot{\theta} (\cos\theta \hat{i} + \sin\theta \hat{j}) \\ &\quad + l\ddot{\theta} (\cos\theta \hat{j} - \sin\theta \hat{i})] dl \\ &= \frac{m}{L} \int_0^L [-\cancel{l^2 \ddot{\theta} \cos\theta \sin\theta \hat{k}} + l^2 \ddot{\theta} \cos^2\theta \hat{k} + \cancel{l^2 \ddot{\theta} \sin\theta \cos\theta \hat{k}} \\ &\quad + l^2 \ddot{\theta} \sin^2\theta \hat{k}] dl \end{aligned}$$

$$\begin{aligned}
 \vec{H}_O &= \frac{m}{L} \int_0^L x \ddot{\theta} x dx \hat{k} \\
 &= \frac{m}{L} \cdot \frac{L^3}{3} \ddot{\theta} \hat{k} \\
 &= \boxed{\frac{m}{3} L^2 \ddot{\theta} \hat{k}}
 \end{aligned}$$

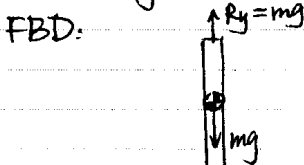
7.93

a) Swinging rod with mass



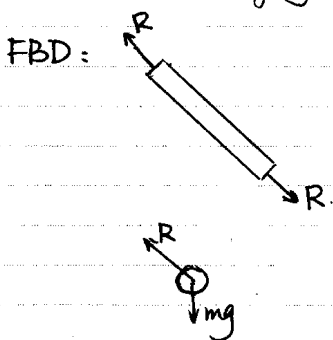
It's not a two force member because the body has mass and accelerates.

b) Stationary rod with mass



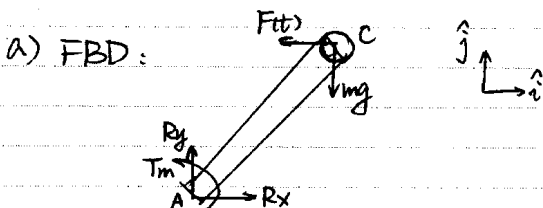
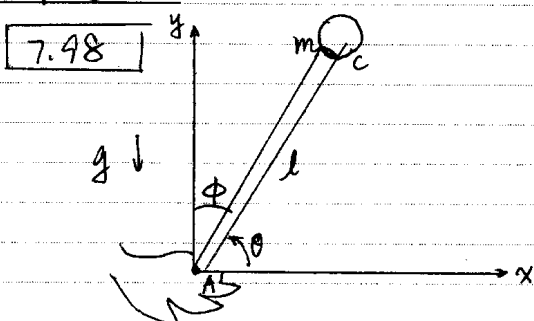
It's a two force member. — the forces are equal and opposite.

c) Massless Swinging rod.



If one includes the ball, the system does not form a two force member (like part a).

If one looks only at the rod (massless), it's a two force member.



b) AMB $\sum \vec{M}_A = \vec{H}_A$

$$\begin{aligned}\sum \vec{M}_A &= \vec{r}_{CA} \times (-mg\hat{j}) + \vec{r}_{CA} \times F_{ct}(-\hat{i}) + \vec{T}_m \\ &= l(\sin\phi\hat{i} + \cos\phi\hat{j}) \times (-mg\hat{j}) + l(\sin\phi\hat{i} + \cos\phi\hat{j}) \times F_{ct}(-\hat{i}) \\ &\quad + T_m\hat{k} \\ &= -mgl\sin\phi\hat{k} + F_{ct}l\cos\phi\hat{k} + T_m\hat{k}\end{aligned}$$

$$\begin{aligned}\vec{H}_A &= \sum \vec{r}_{A} \times \vec{a}_C m_C \\ &= l(\sin\phi\hat{i} + \cos\phi\hat{j}) \times [l\ddot{\phi}(-\sin\phi\hat{i} - \cos\phi\hat{j}) - l\dot{\phi}(\sin\phi\hat{j} - \cos\phi\hat{i})]m \\ &= l^2\ddot{\phi}(-\sin\phi\cos\phi + \sin\phi\cos\phi)\hat{k} m \\ &\quad - \cancel{m} l^2\dot{\phi}(\sin^2\phi + \cos^2\phi)\hat{k} \\ &= -ml^2\ddot{\phi}\hat{k}\end{aligned}$$

$$\therefore \{ \} \hat{k}: ml^2\ddot{\phi} = mgl\sin\phi - Fl\cos\phi - T_m$$

$$\therefore \boxed{\ddot{\phi} - \frac{g}{l}\sin\phi + \frac{F}{ml}\cos\phi + \frac{T_m}{ml} = 0} \quad (*)$$

c) Say, T_m can be any function of ϕ and $\dot{\phi}$
 Guess, T_m should be in such a form that makes equation (*) looked like:

$$\ddot{\phi} + c\dot{\phi} + k\phi = 0$$

This is the equation of a damped vibration, and so the pendulum should be driven to $\phi=0$, stay upright.
 Alternatively, you can choose the torque T_m to make equation (*) even simpler: $\frac{T}{ml} = \frac{g}{l}\sin\phi + c\dot{\phi}$

Then eqn (*) looks like:

$$\ddot{\phi} + c\dot{\phi} = 0 \quad (\text{Assume } F_{ct}=0)$$

d). Say, $T_m = ml^2(c\dot{\phi} + k\phi)$ $c > 0, k > 0$, constant.

$$\ddot{\phi} - \frac{g}{L} \sin \phi + c\dot{\phi} + k\phi = 0$$

linearize the eqn: when small angle, $\sin \phi \approx \phi$

$$\ddot{\phi} + c\dot{\phi} + (k - \frac{g}{L})\phi = 0$$

we assume the solution of the form

$$\phi = Ae^{\lambda t}$$

$$\therefore \lambda^2 + \lambda c + (k - \frac{g}{L}) = 0$$

$$\therefore \lambda = \frac{-c \pm \sqrt{c^2 - 4(k - \frac{g}{L})}}{2}$$

If $c^2 > 4(k - \frac{g}{L})$ $\phi = A_1 e^{\lambda_1 t} + A_2 e^{\lambda_2 t}$ where $\lambda_1, \lambda_2 < 0$

If $c^2 = 4(k - \frac{g}{L})$ $\phi = (A_1 + A_2 t) e^{-\frac{c}{2}t}$

If $c^2 < 4(k - \frac{g}{L})$ $\phi = C \sin(\sqrt{4(k - \frac{g}{L}) - c^2}t + \psi) e^{-\frac{c}{2}t}$

All of them show that ϕ will decay finally.

e) For the nonlinear equations,

$$\ddot{\phi} - \frac{g}{L} \sin \phi + c\dot{\phi} + k\phi = 0$$

we implement a matlab program to see how the angle ϕ is changed with time.

```
% program for Problem 7.98 e)
```

```
global g L m c k ;
```

```
g=10; L=1; m=1; c=2; k=20;
```

```
tspan=[0 10];
```

```
Fi0=pi/3; Fidot0=0; % initial conditions
```

```
z0=[Fi0 Fidot0];
```

```
[t,z]=ode45('deriv11',tspan,z0);
```

```
Fi=z(:,1); Fidot=z(:,2);
```

```
plot(t,Fi);
```

```
hold on;
```

```
[t,z]=ode45('deriv12',tspan,z0);
```

```
fil=z(:,1); fidot=z(:,2);
```

```
plot(t,fil,'--');
```

```
function zdot=deriv11(t,z);
```

```
global g L m c k;
```

```
Fi=z(1);
```

```
alpha=z(2);
```

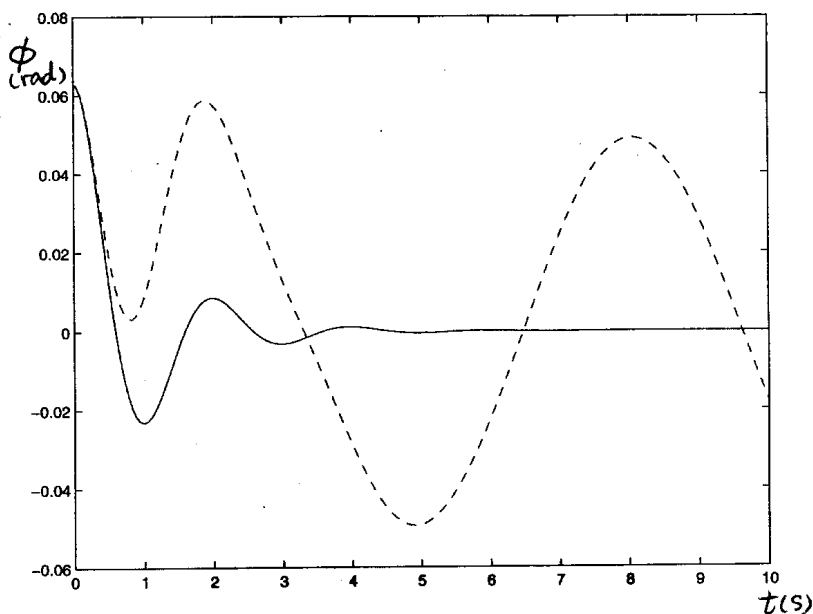
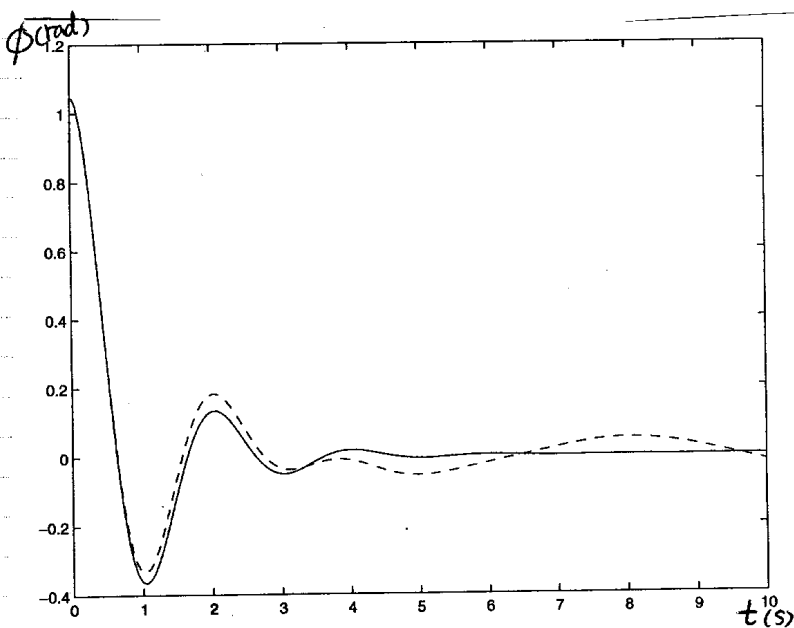
```
alphadot=+g/L*sin(Fi)-c*alpha-k*Fi;
```

```
zdot=[alpha alphadot];
```

```

function zdot=deriv12(t,z);
global g L m c k;
Fi=z(1);
alpha=z(2);
alphadot=+g/L*sin(Fi)-c*alpha-k*Fi+0.5*sin(t)*cos(Fi);
zdot=[alpha alphadot]';

```



f) Suppose $F(t) = 0.5m \sin t$. The plot of ϕ vs. time is shown as above. As you can see, at the beginning, disturbing force won't affect much. But ~~later~~ later, it becomes dominant.